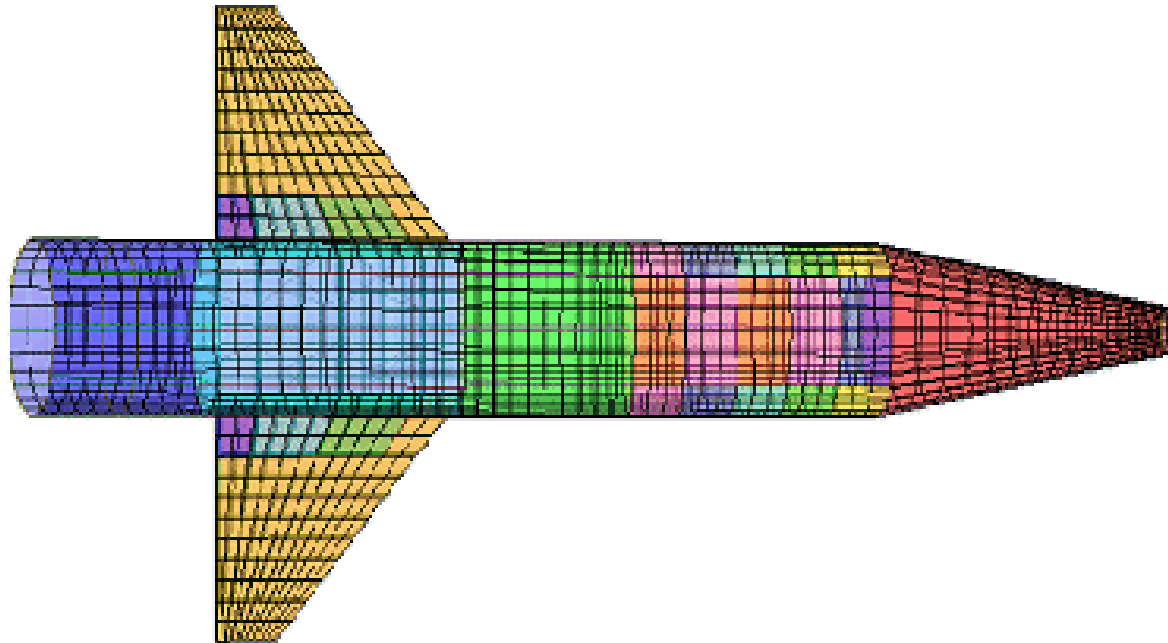




Analysis Approach



Thermoelastic Formulations: Objective



Compute very accurate stress/strain fields throughout a panel or beam depth for general mechanical or thermal loading conditions. The loadings may be either user-defined or from finite element solutions. Handle any general panel concept, shape, and composite layup very rapidly. Include coupling effects of panel stiffeners.

Thermoelastic Formulations: Technical Approach



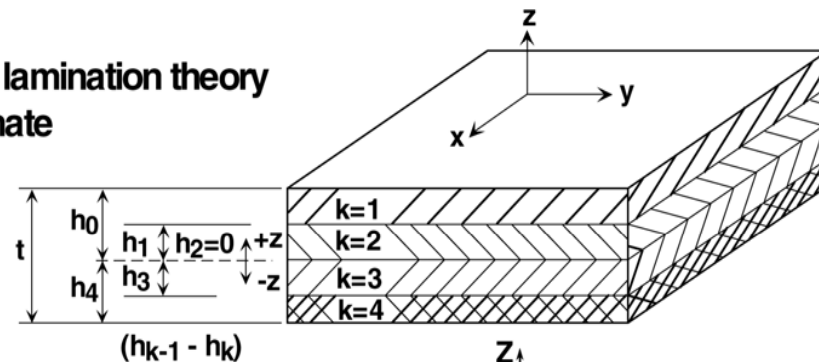
1. Formulate the full membrane, bending, and membrane-bending coupling terms in a general way for any panel concept
2. Supplement the general formulation with additional panel or beam concept specific formulations
3. Use the formulations to permit general loadings and general boundary conditions on panel edges. Permit uniform pressures and through-the-thickness temperature gradients on the panel surface.
4. The general loadings can be user specified or come from FEA

Thermoelastic Stiffened Panel Formulation (General Behaviors)

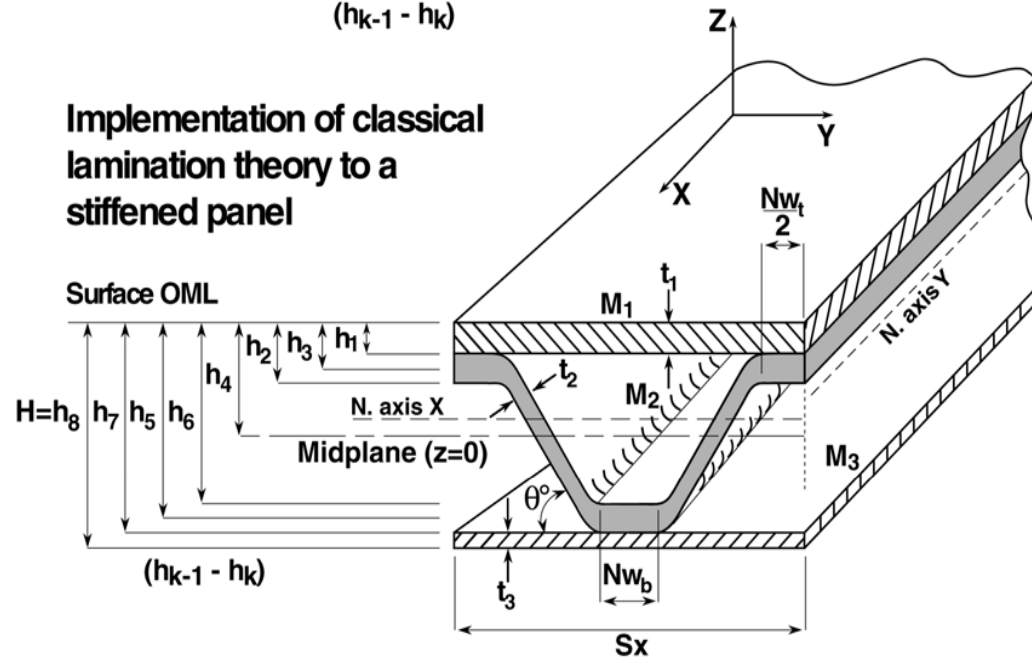


- Classical Lamination Theory is Extended to represent any stiffened cross sectional shape
- General panel behaviors, are quantified with:
 - Stiffness terms $[A]$, $[B]$, $[D]$
 - Thermal coefficients $[A^\alpha]$, $[B^\alpha]$, $[D^\alpha]$

Classical lamination theory of a laminate



Implementation of classical lamination theory to a stiffened panel

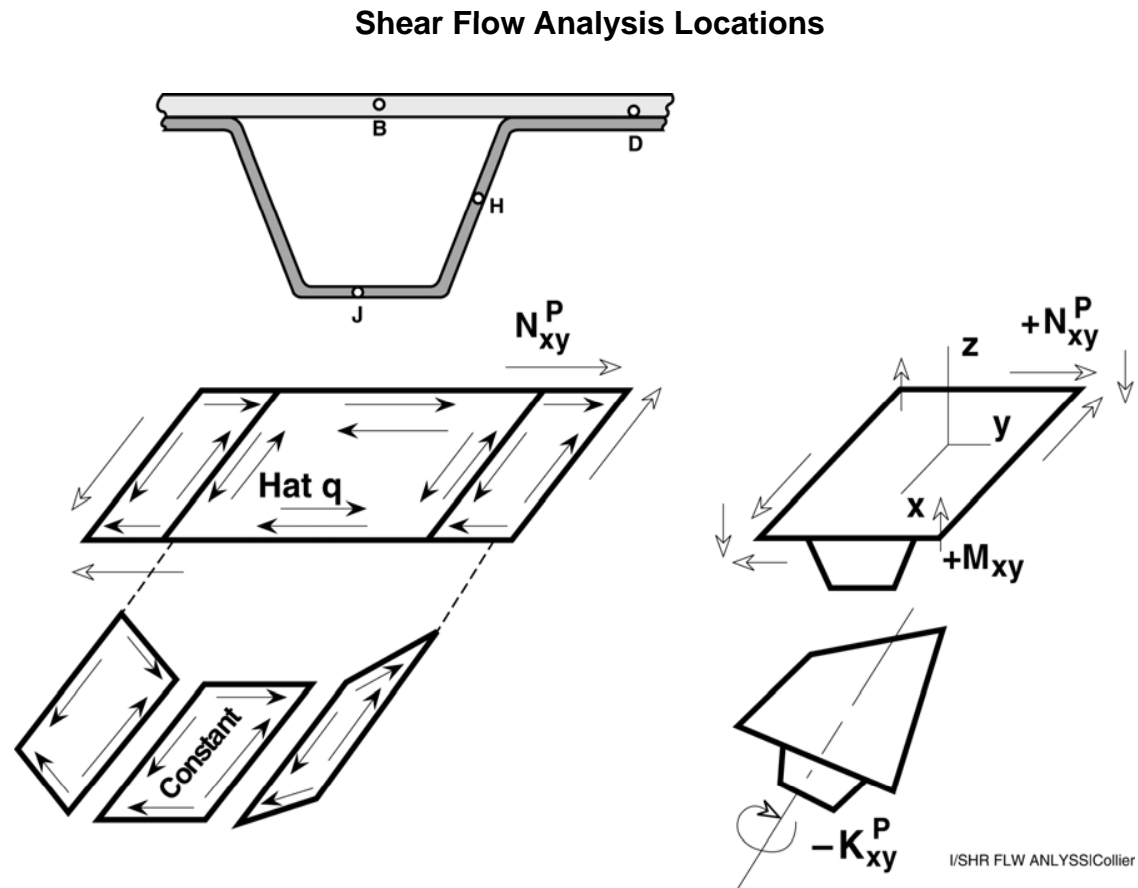


I/LMNT FRMLTN(2)vertCollier

Thermoelastic Stiffened Panel Formulation (Unique Behaviors)



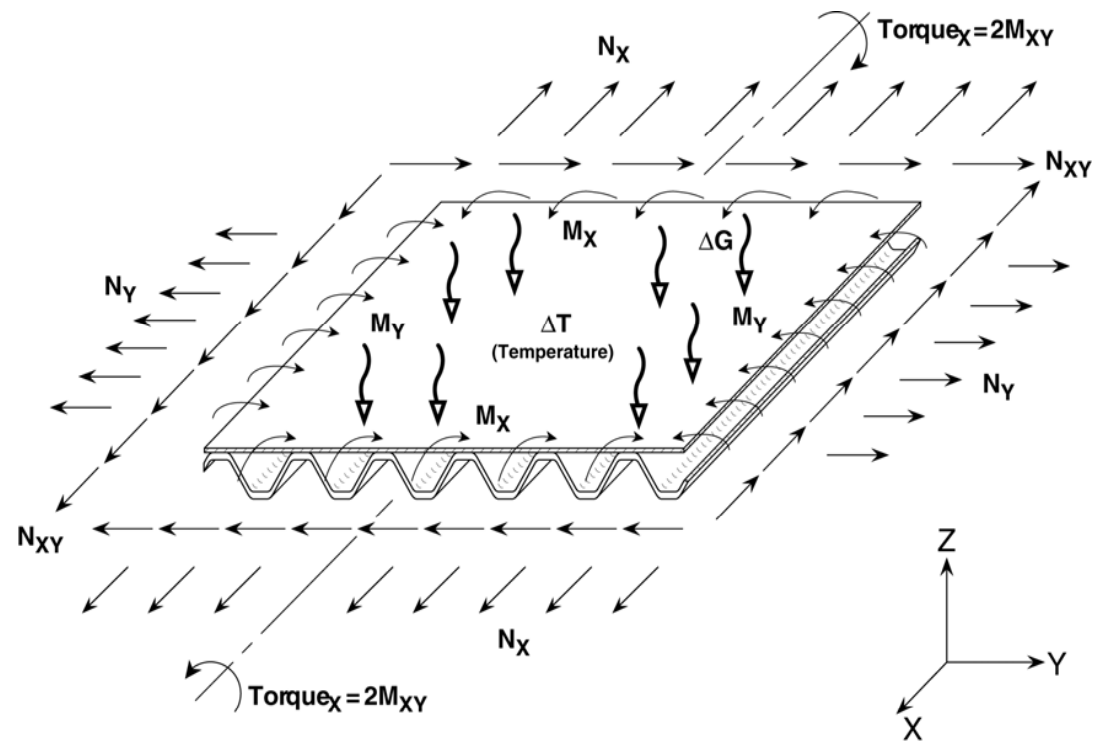
- Each panel concept has additional unique characteristics to quantify
- For example, hat stiffened panels have shear flow in the closed section to account
- The closed section adds considerable bending-twisting stiffness which increases panel stability
- Unique behaviors are quantified without classical lamination theory



Balanced Free Body Diagram (FBD) Approach



- All solutions reduce to a balanced free body diagram approach, regardless if the loading and/or boundary conditions come from:
 - FEA
 - user defined input
- Consistently applied thermoelastic formulations guarantee
 - equilibrium of forces

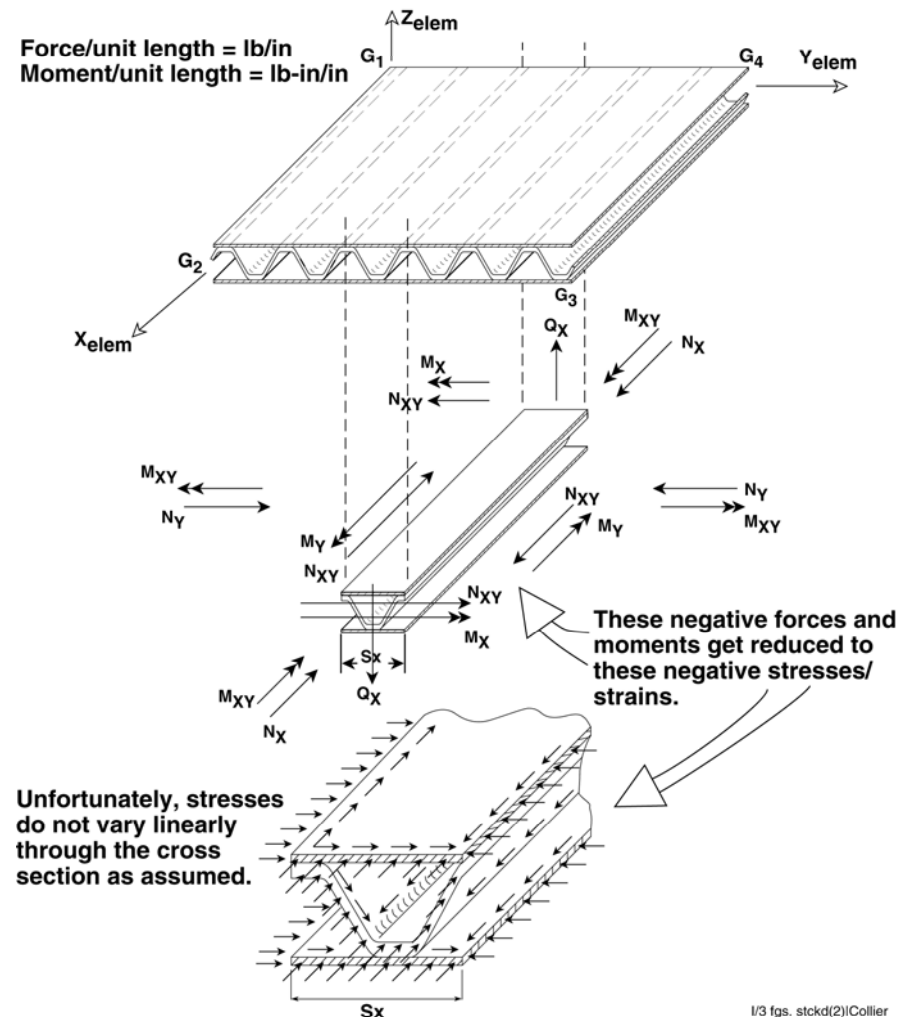


I/ST-SZICollier

HyperSizer overcomes traditional analytical approach shortcomings



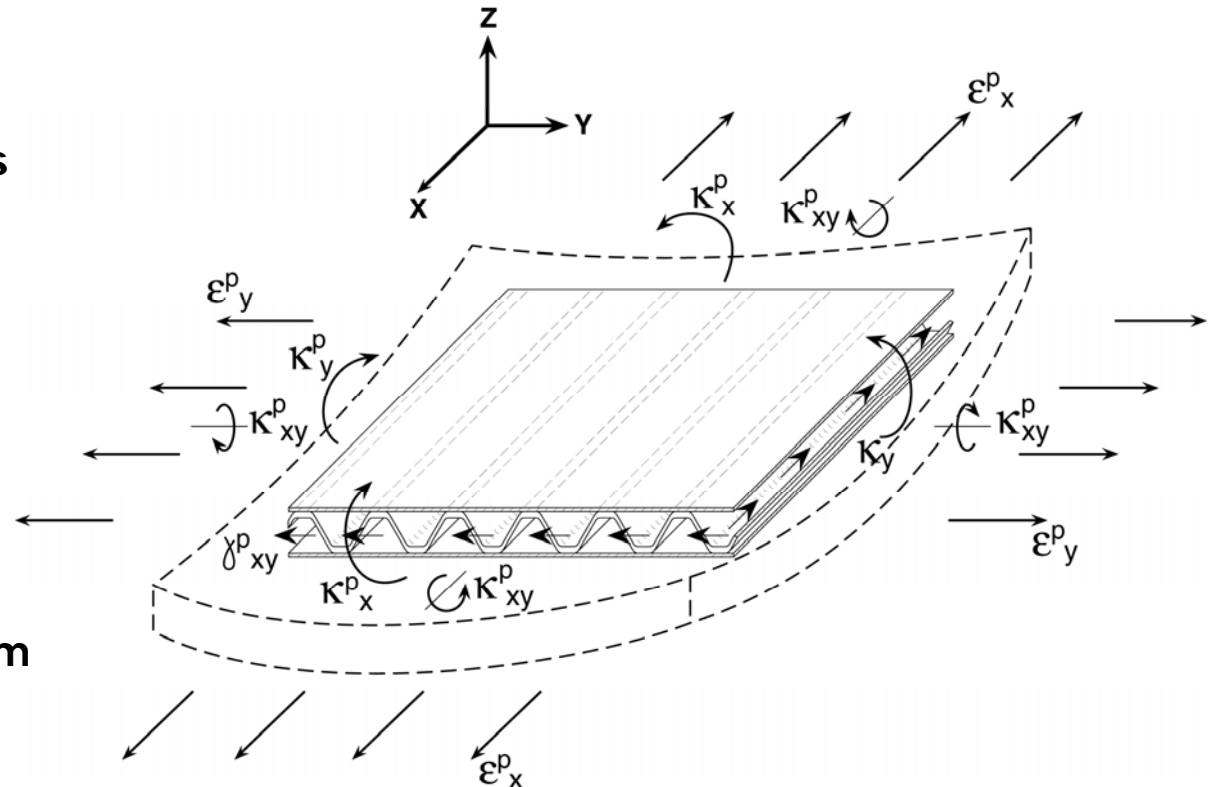
- Depicted in the figure is a wide strip approach with forces applied to obtain stresses directly
- Traditional approaches mistakenly assume stresses vary linearly through the depth
- Only strains vary linearly through the panel depth



Panel Reference Plane Strains and Curvatures Are Computed from FBD Forces

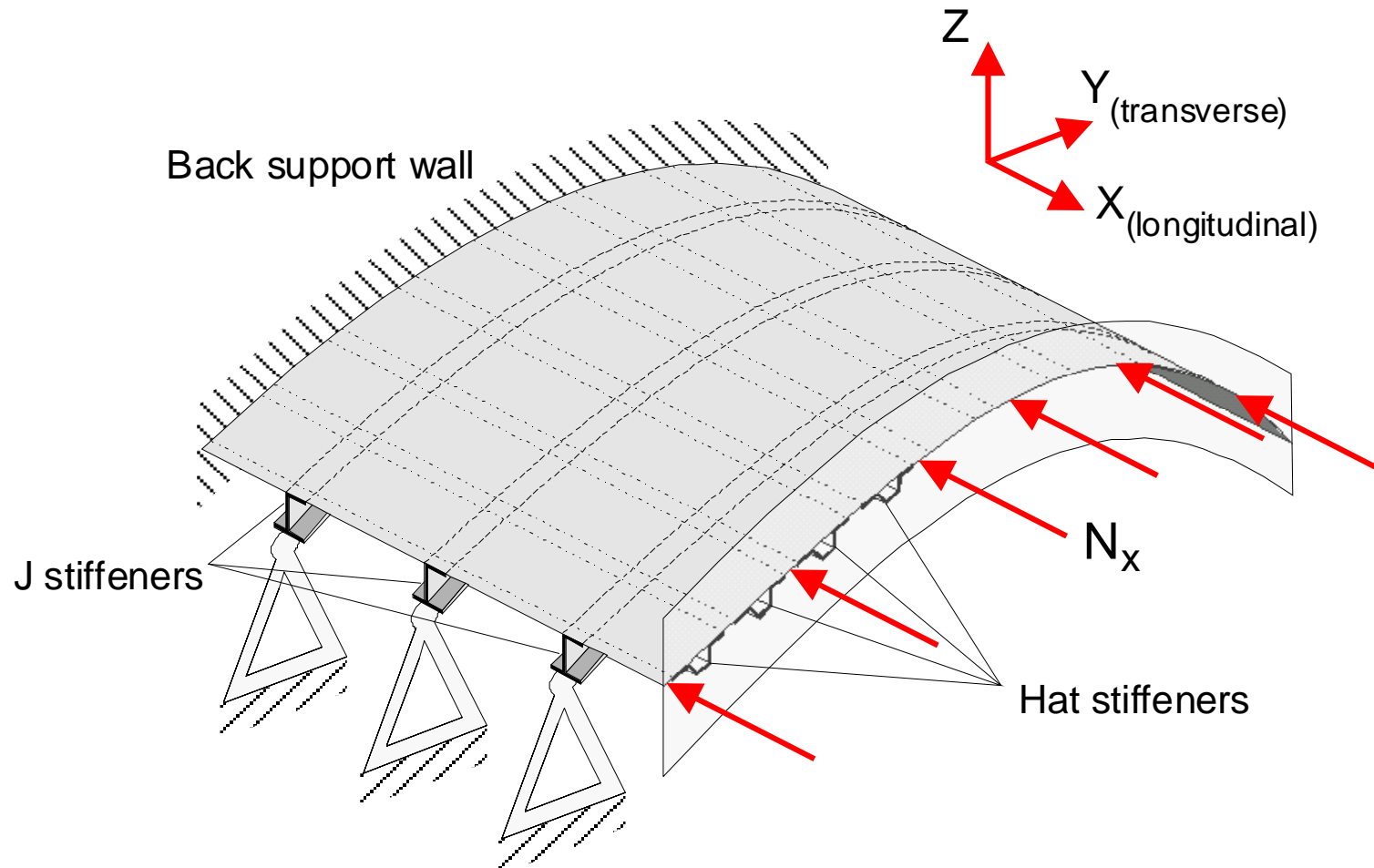


- Matrix math approach is used, which includes all panel response couplings
- Consistently applied thermoelastic formulations guarantee
 - strain compatibility
- Ply strains, throughout the panel depth, are accurately computed from the panel response
- ply stresses are calculated from ply strains



I/SI(lrg secbcklmg)(2)Collier

Without FEA; Uniform Compression Load and Enforced Displacement-Hat Panel

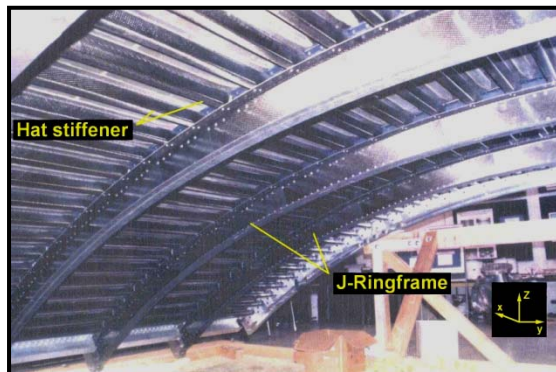


1st Use: Accurate Analysis of Many Potential Failure Modes

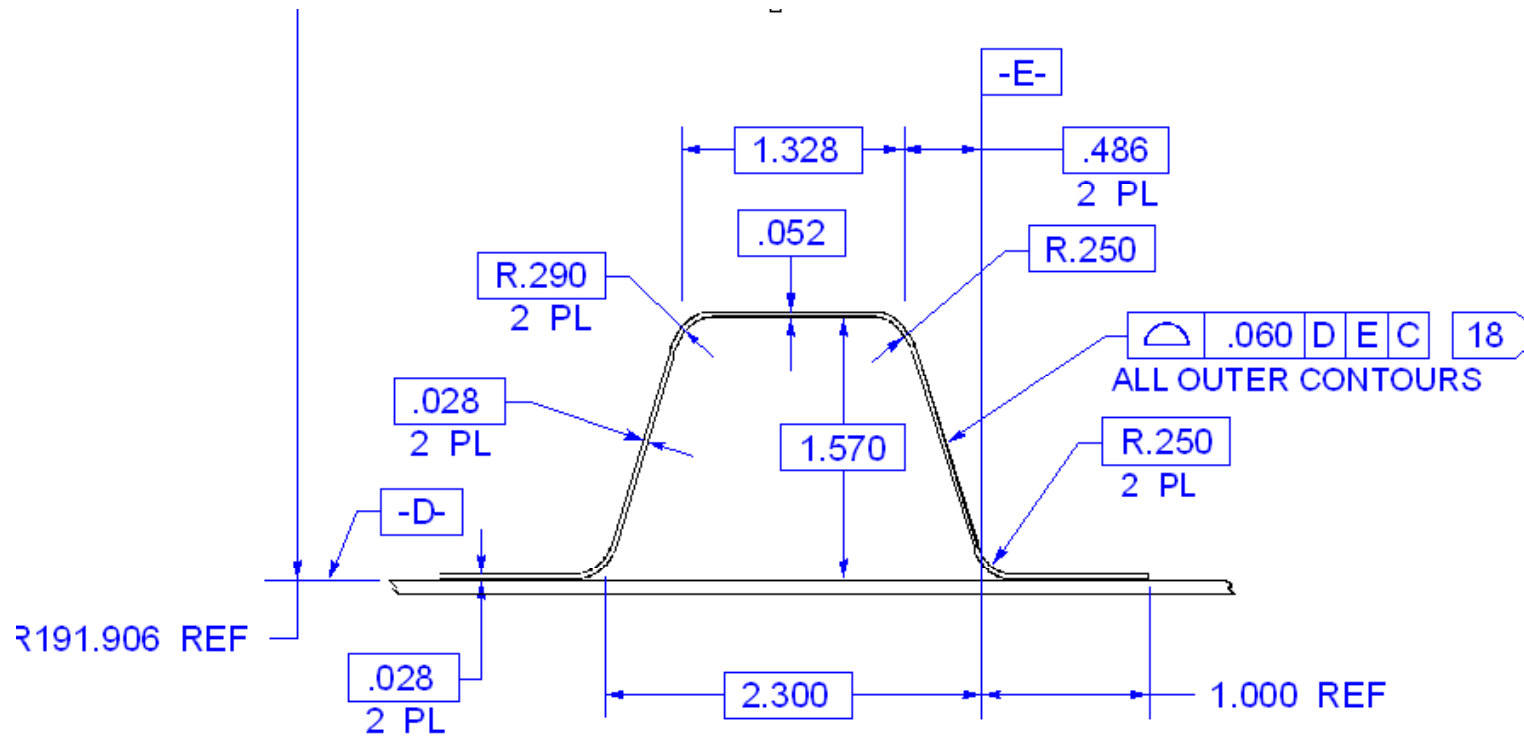


RLV Intertank Failure
Load Prediction for a
NASA Langley Test

Analyze a structural panels and beams in *detail*

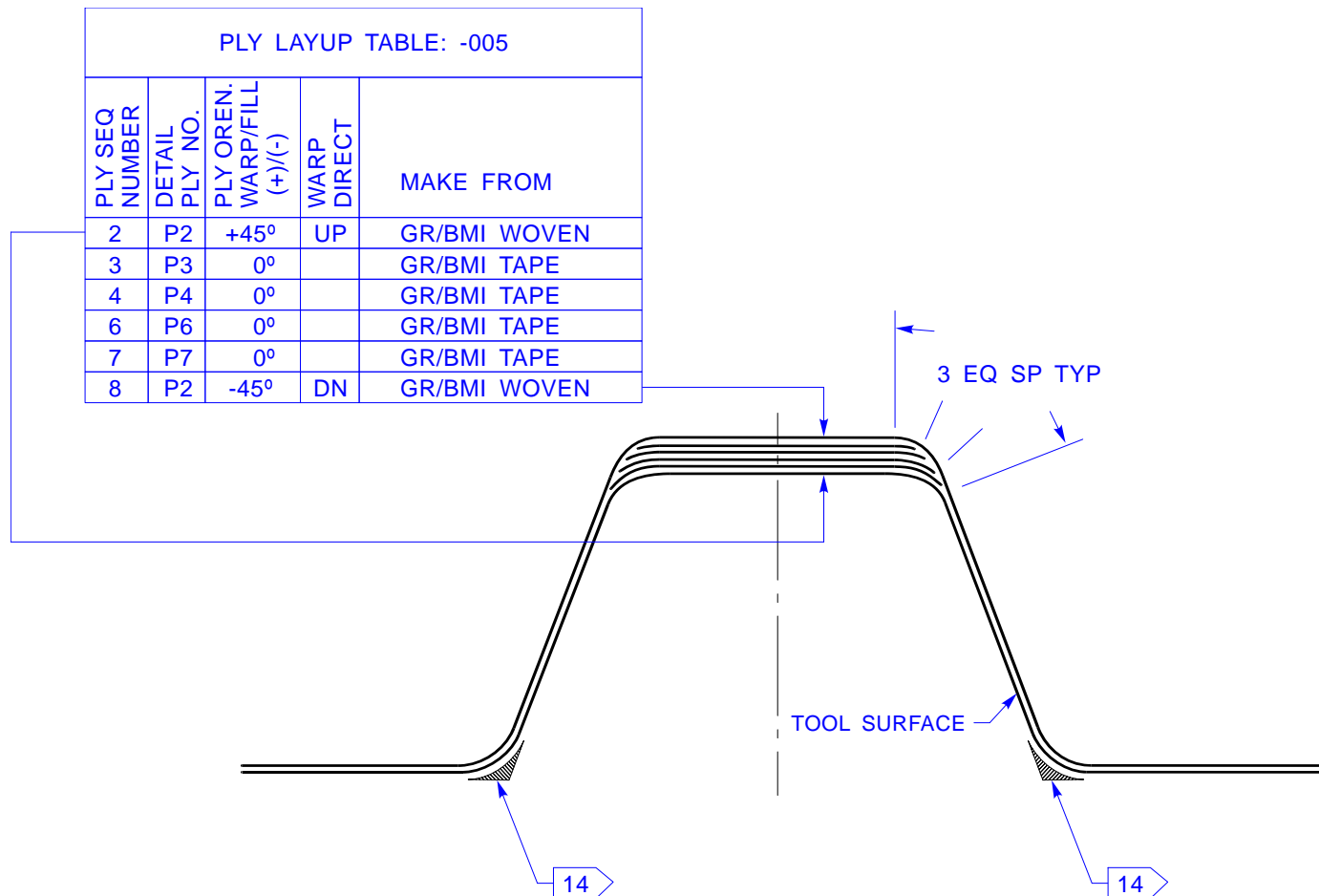


Cross Section Dimensions



VIEW D-D E36
SCALE 1/1

Coresheet Layup Schedule, Hybrid Woven and Tape Forms, Ply drop-offs in the Crown



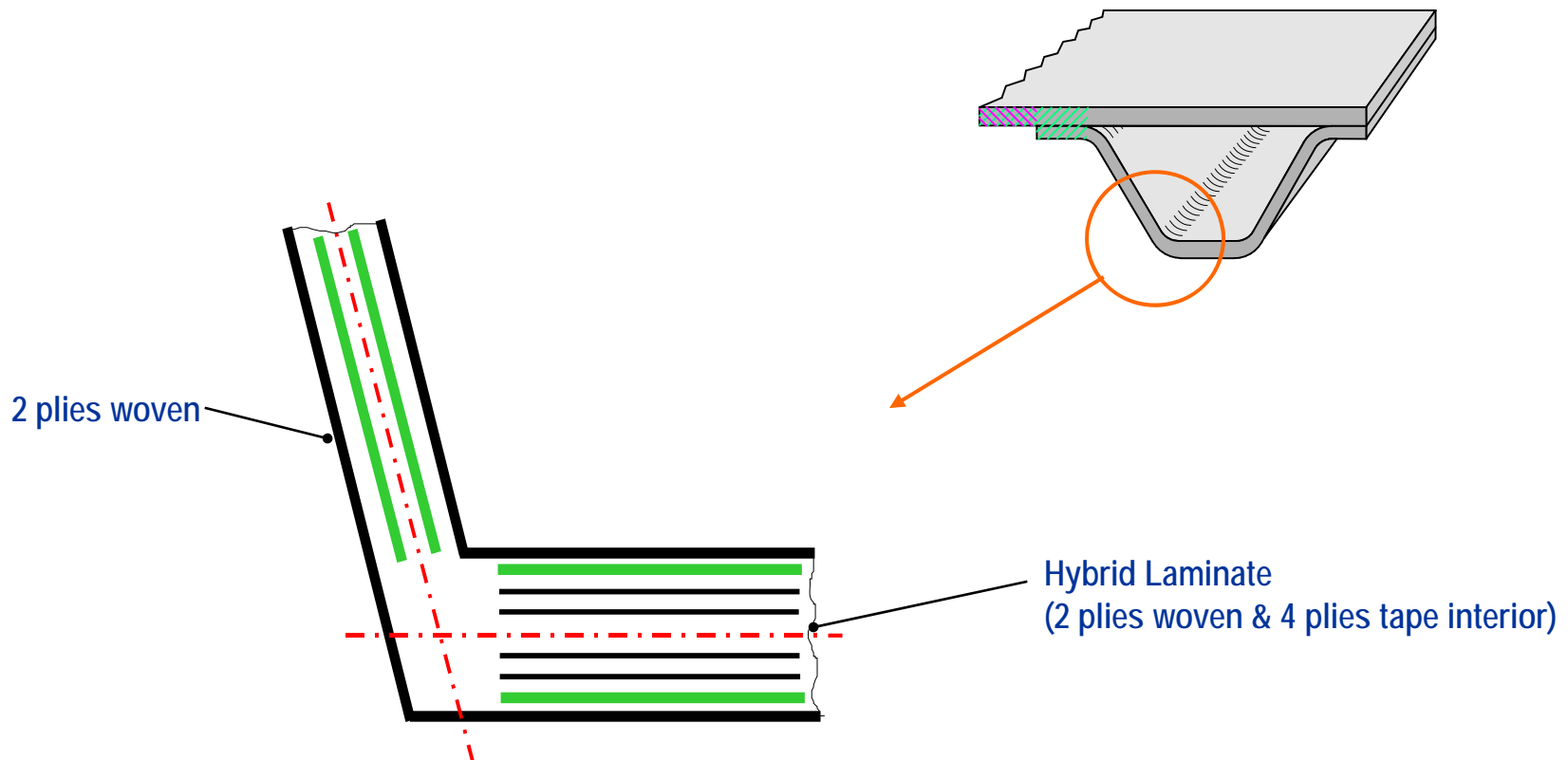
SECTION R-R E36

PLY SCHEMATIC FOR -005
NO SCALE

Formulation Accuracy Checks



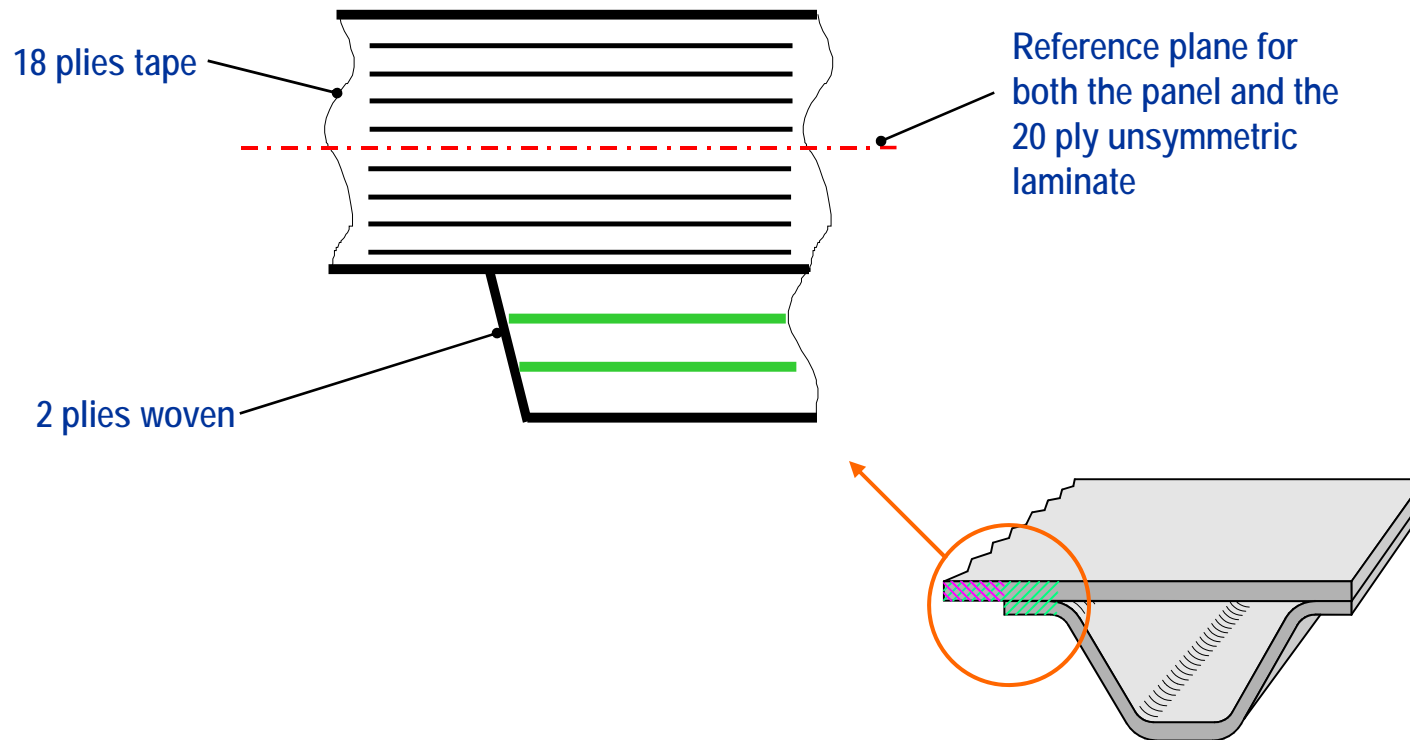
Hybrid Laminate of the Hat Crown



Formulation Accuracy Checks



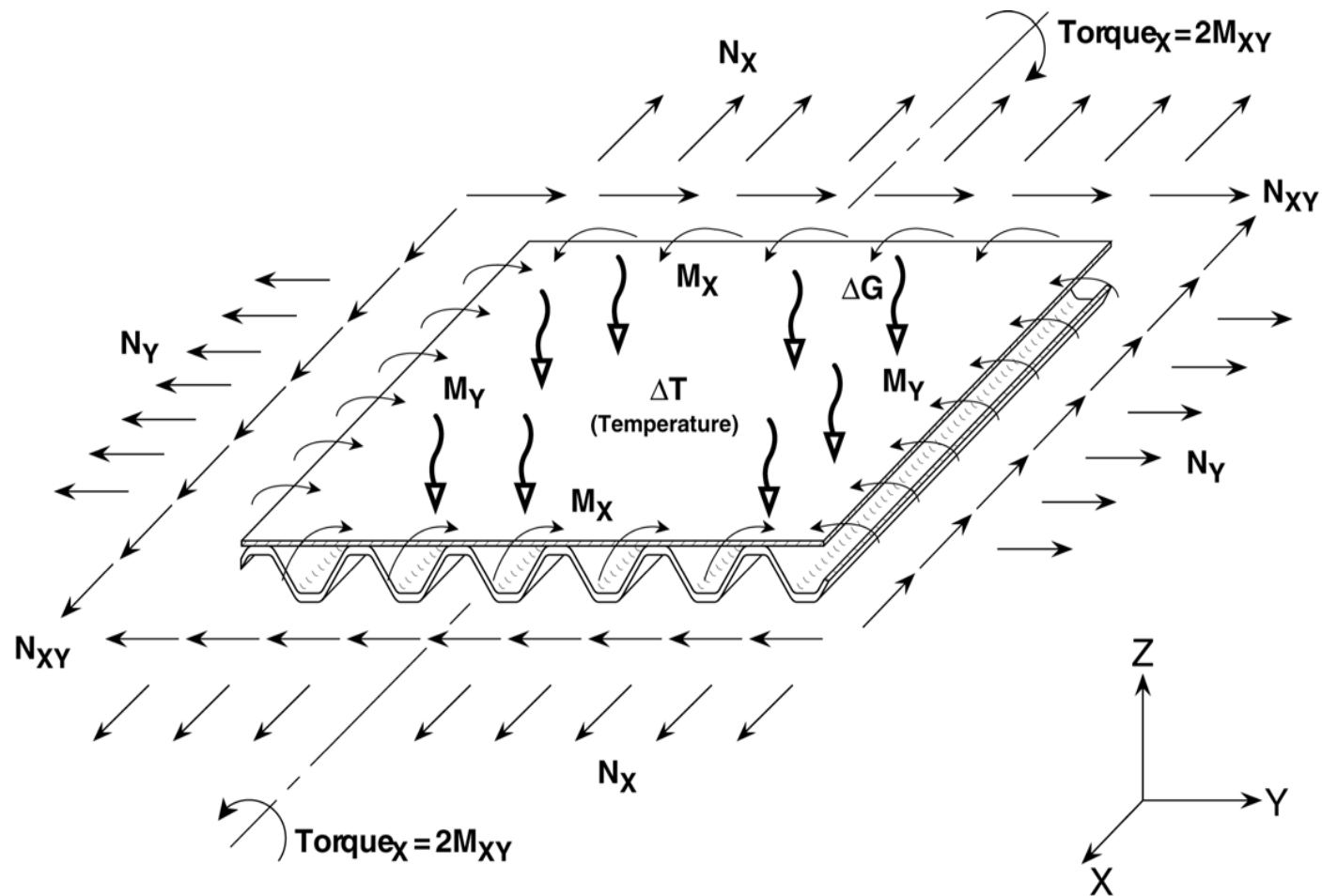
Skin - Stiffener Bonded Laminate



Formulation Accuracy Checks



Balanced Free Body Diagram (FBD) & Strain Compatibility Approach



I/ST-SZICollier



Balanced FBD Implementation of General Edge Loadings and Boundary Conditions



Concepts
Design-to Loads
Failure
Buckling
Notes

Variables
FBD
Object Loads
Computed Properties
Options

Input (Per Load Case)

ULTIMATE-MECHANICAL Load Case #5 "Mechanical Load Set #104 (Run Deck #1) "Wing and Tank Pressure", Thermal None." (Mechanical Set #104, No Ther

Mechanical Load Set #104 "Wing and Tank Pressure" Ref Temp Temp

No Thermal Load Set Pressure TT Grad

FEA Loads Applied Unit Value

User Loads For Strength Analysis

For Buckling Analysis

	N_x, ϵ_x	N_y, ϵ_y	N_{xy}, γ_{xy}	M_x, κ_x	M_y, κ_y	M_{xy}, κ_{xy}	Q_x	Q_y
	Load	Load	Load	Constrained	Constrained	Constrained	Load	Load

Superimposed Loads

Panel Pressure Beam-Column Moments

Zero Out FEA Computed Moments FIXED Boundary Condition

	M_x	M_y	Q_x	Q_y
MidSpan	0	0	0	0
EdgeCntr	0	0	0	0

Point Free Body Diagram (Constant Forces)

a (X length)

b (Y length)

Free Body Diagram Output (Controlling Factored Loadcase)

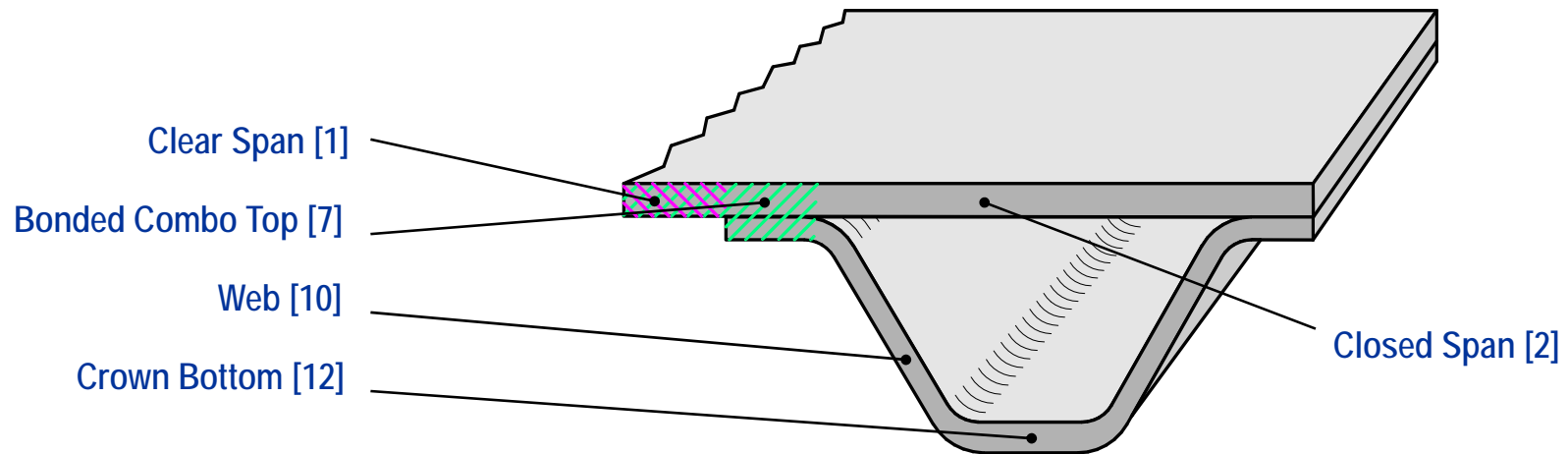
Controlling Analysis Load: STRENGTH

	N_x, ϵ_x	N_y, ϵ_y	N_{xy}, γ_{xy}	M_x, κ_x	M_y, κ_y	M_{xy}, κ_{xy}	Q_x	Q_y
Virtual Loads								
Design-to Loads	-2.7525	-647.942	-814.199	-4.42358	-442.728	-451.675	-59.3113	-70.5098
Design-to Deformation	4.140223E-04	-1.101663E-03	-3.417936E-03	2.847224E-05	-1.025333E-04	6.513386E-04		

Object Loads



Load Results						
Object	Nx	Ny	Nxy	Mx	My	Mxy
Clear Span [1]	-1534.22	-567.892	16.8327	-1.15463E-14	-1.11022E-14	8.88178E-16
Closed Span [2]	-1534.22	-567.892	13.2117	-1.68754E-14	-1.02141E-14	4.44089E-16
Bonded Combo Top [7]	-1925	-567.892	16.8327	-23.9166	-18.0217	0.343983
Web [10]	-169.591	1.84627E-13	3.62103	0	0	2.6177E-15
Crown Bottom [12]	-1058.52	1.77636E-13	3.62103	-3.33067E-15	-7.49401E-16	5.99867E-15



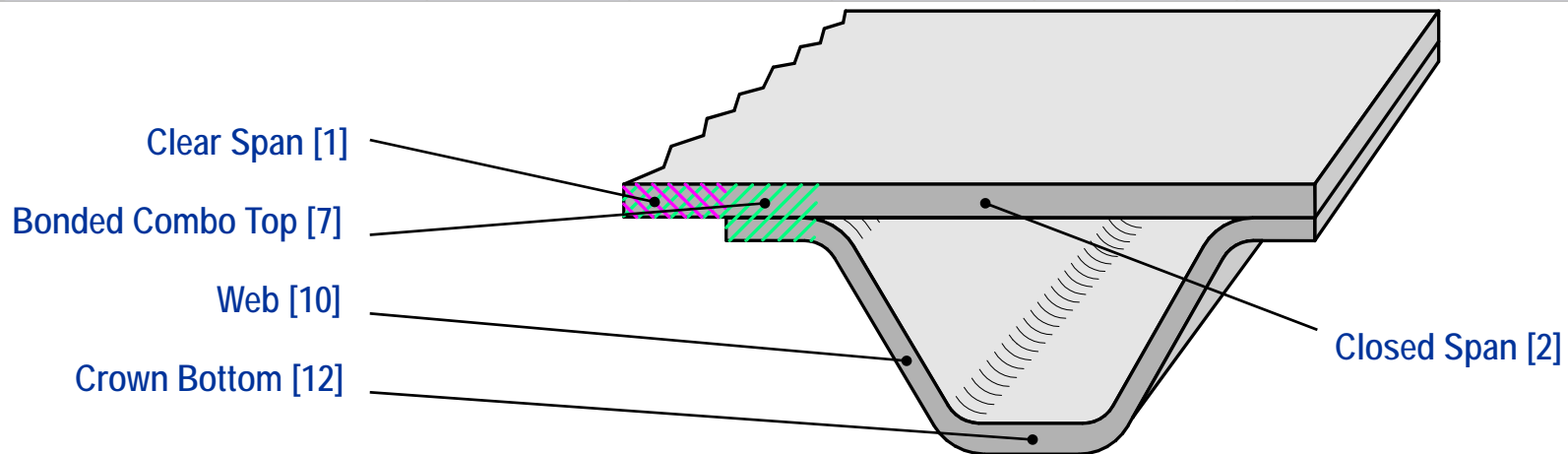
Formulation Accuracy Checks



Summation Verification of the Object Membrane Forces



Load Results						
Object	Nx	Ny	Nxy	Mx	My	Mxy
Clear Span [1]	-1534.22	-567.892	16.8327	-1.15463E-14	-1.11022E-14	8.88178E-16
Closed Span [2]	-1534.22	-567.892	13.2117	-1.68754E-14	-1.02141E-14	4.44089E-16
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Crown Bottom [12]	-1058.52	1.77636E-13	3.62103	-3.33067E-15	-7.49401E-16	5.99867E-15



Ply integrated stresses = object loads ; Summation of object loads = applied loadings

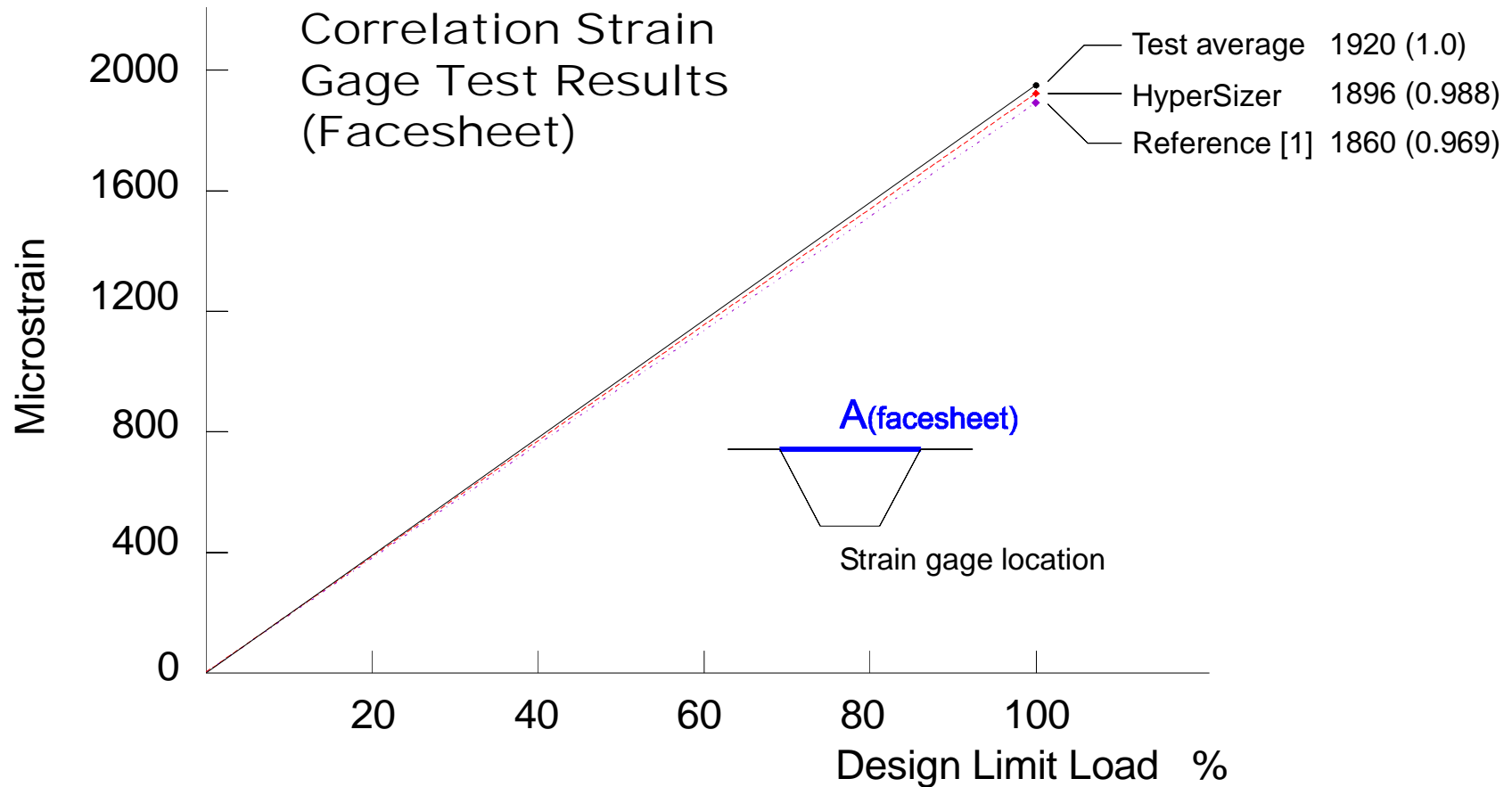
FBD balance and strain compatibility maintained for each individual object and for the entire panel

Failure Analyses



- **Material strength, panel buckling, local buckling, crippling, deformation, stiffness, and natural frequency**
- **Each failure analysis is summarized with a margin-of-safety (MoS) for limit and ultimate loads**
- **Each failure analysis can be turned on or off independently for specific components or entire projects**
- **Company administrator can change product default analyses, users can change project defaults**

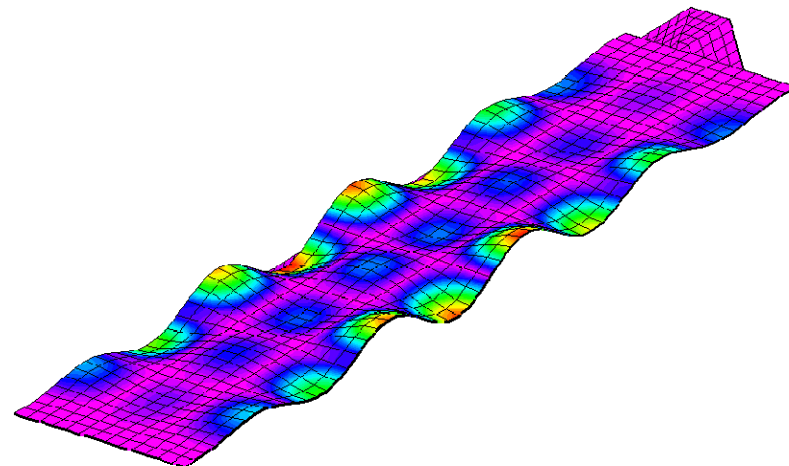
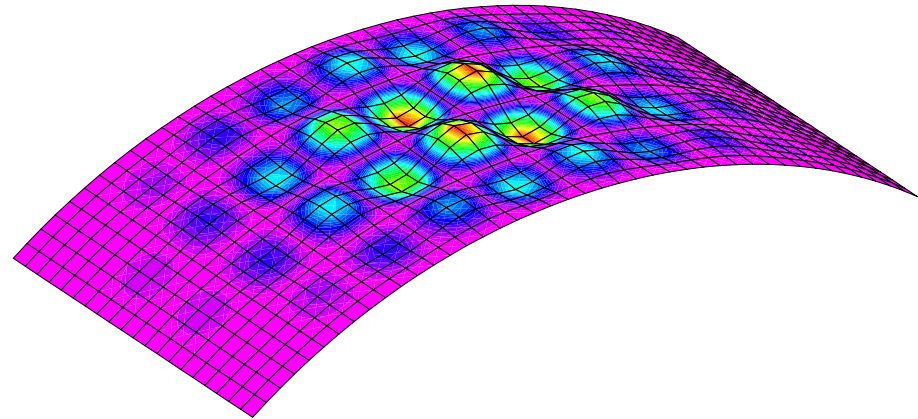
Pretest Prediction to Uniform Compression Load – Hat Stiffened Panel Structural Certification



Global to Local Analyses are Performed Automatically, All the Time



- HyperSizer performs **panel** buckling every time---->
- HyperSizer performs **local** buckling every time---->
- HyperSizer performs **all** analyses every time



Failure Analyses



Important Observations About the Test Correlation Process



The aerospace industry builds many FEMs to capture different responses of the structure

a model for system buckling

a model for panel buckling

a model for local buckling

a model for detailed stresses and material strength

For this test, at least four non-HyperSizer models were made

HyperSizer needs just one coarse planar FEM